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Performance enhancement of Waffles (Appalam) drying using mixed mode solar dryer

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Abstract

In this present work, investigations are made to study the performance of mixed mode solar dryer for waffles drying. The design, development and performance analysis of mixed mode solar dryer for drying of waffles (appalam) is reported. The design was developed under metrological conditions at Puducherry. A novel numerical model was developed to enhance the productivity rate by reducing the drying time of the waffles (appalam).

Keywords: Mixed mode solar dryer, Productivity Rate, Waffles, Analysis

1. Introduction

The main source of all capital for renewable energy is solar energy. To solve current energy crises, improvement in sustainable solar energy technologies to harness the tremendous power of the sun is required. Solar power has tremendous potential to meet the rising energy needs of the growing population. Among the rural farmers of developing countries, solar drying technology can be easily adopted. In the case of waffles (appalam) drying, the drying process is of particular importance, since they are especially susceptible to insects.

1.1 Solar energy

The earth's surface has received approximately 1014 KW of solar energy from the sun. The energy equivalent of around 1 HP or 1 KW is supplied by one square metre of land exposed to direct sunlight. Large-scale conversion of solar energy involves a large investment of resources. In contrast to other sources of electricity, solar energy

has some strong advantages. Planet has obtained from the sun approximately 1014 KW of solar energy. Approximately 1 HP or 1 KW of energy equivalent is given. In contrast to other sources of electricity, solar energy has some strong advantages. Solar radiation does not contaminate the atmosphere or jeopardise environmental equilibrium. It prevents serious problems such as mining, extraction and transportation.[1-5]

1.2 Solar drying

Sample Drying, long before frying, was probably the first food preservation tool ever used by man. The appropriate food preservation technology for solar dryers. The earliest method of drying farm produce ever known to man is "Sun drying". It includes simply laying the farm products on mats, roofs or drying floors in the sun. The approach often involves vast areas of land that take time and are highly labour intensive. Solar dryers are specialized instruments in the solar drying process that control the drying process. They protect agricultural produce from insect pests, dust and

rain damage. Higher temperatures, lower relative humidity and lower product moisture content are produced by solar dryers. Solar dryers can be considered one of the solutions to the food and energy shortages of the planet. Solar dryers are mainly divided into three groups, which are

- ✓ Direct solar drying
- ✓ Indirect solar drying
- ✓ Mixed mode solar drying

1.3 Direct solar drying

A solar cabinet dryer is also regarded as the working theory of direct solar crop drying. The air entering the cabinet from below and exiting through the top exit takes away the moisture here. A percentage of the overall solar radiation that affects the glass cover is reflected back into the atmosphere and the remainder is transmitted within the cabinet.

1.4 Indirect solar dryer

Inside an opaque drying cabinet, the crops in these indirect solar dryers are located in trays or shelves. The warm air is allowed to flow through the wet crop and over it. Owing to the distinction of moisture concentration, drying takes place.

1.5 Mixed mode solar dryer

The solar dryers combine the characteristics of direct and indirect solar energy dryers of the mixed-mode kind. In both the flat plate air heater and the drying chamber, the storage of solar energy takes place. Solar energy will also be obtained by the outer portion of the dryer.

For domestic purposes, step-type solar dryers are widely used. Solar tunnel dryers are typically used in manufacturing industries.

2. Experimental arrangements

The mixed-mode has a pre-heating mechanism for the air via a solar air heater. Food items are respectively heated by hot air and direct exposure to solar radiation from the air heater. The mixed solar dryer mode consists primarily of four components they are,

- ✓ Flat plate collector
- ✓ Blower
- ✓ Drying chamber
- ✓ Thermocouple

A flat-plate solar air heater and a drying chamber were part of the laboratory model. The collector dryer assembly consists of a 1 mm aluminium sheet

of smoke black coating used as a solar radiation absorber board.[6-10]

2.1 Flat plate collector

A flat-plate collector (FPC) is a device for collecting and converting solar energy into thermal energy. It is the basis of solar thermal systems in a medium temperature range for many applications. A traditional solar air heater typically consist of an absorber plate with a parallel plate below providing a narrow passage into which the air to be heated flows.

2.2 Blower

It is a device that is used and sent through the ambient air to suck the air from the surrounding.

2.3 Drying chamber

The gross solar dryer scale is 400x330x330mm. With the assistance of sheet metal, the blower is paired with the solar air heater. For a certain thickness and glass cover, the dryer is covered with wood.

2.4 Thermocouple

A thermocouple is a device used to measure temperature accurately. It is often made of two dissimilar metals. A device in which a temperature difference is deduced from a thermoelectric potential calculation at the ends of a pair of separate metal wires. The presence of a temperature gradient in a metal or alloy results in the setting up of an electrical potential gradient along the temperature gradient.

2.5 Working fluid

Air is used as working fluid. The properties of air at 30°C

Density = 1.165Kg/m³

Specific heat capacity = 1005J/Kg-K

Thermal conductivity = 0.02675W/m-k

Kinematic viscosity = 15.06×10⁻⁶ m²/s

Absolute viscosity = 18.63 ×10⁻⁶N-s/m²

Prandtl number = 0.701

3. Design calculation

Determination of Flux Absorbed by Absorber Plate:

For March 21 – at 12:00Pm

$I_g = 950\text{W/m}^2$

Latitude of puducherry (ϕ) = 11°55'

Angle of tilt (β) of solar air heater = 10°+lat (ϕ)

$\beta = 21^\circ 55'$

Declination angle:

$\delta = 23.45\sin [(360/365) \times (284+n)]$

n - Number of days

n = 80

$\delta = -0^{\circ}24'$

$\omega = 15[12-LST]$

$\omega = 0^{\circ}$

$\cos \theta = (\sin \delta \times \sin (\phi-\beta)) + (\cos \omega \times \cos \delta \times \cos (\phi-\beta)) = 0.986$

$\cos \theta z = (\sin \delta \times \sin \phi) + (\cos \omega \times \cos \delta \times \cos (\phi)) = 0.977$

$\omega s = \cos^{-1}(-\tan \phi \times \tan \delta) = 89^{\circ}54' (1.569 \text{ rad})$

$\omega st = \cos^{-1}(-\tan (\phi-\beta) \times \tan \delta) = 90^{\circ}4' (1.572 \text{ rad})$

$R_b = \omega st \sin \delta \sin (\phi-\beta) + \cos \delta \sin \omega st \cos (\phi-\beta) \omega s \sin \delta \sin \phi + \cos \delta \sin \omega s \cos \phi = 1.01$

$R_d = (1 + \cos \beta) / 2 = 0.964$

$R_r = \rho (1 - \cos \beta) / 2$

Assume $\rho = 0.2$

$R_r = 0.0072$

$I_o = 1.367 \times \{1 + 0.033 \cos (360 / 365) \times (n)\} \times (\sin \delta \times \sin \phi) + (\cos \omega \times \cos \delta \times \cos (\phi)) 15$

$I_o = 1.344 \text{ KW/m}^2$ $I_g H_g = I_o H_o (a + b \cos \omega)$

$a = 0.409 + 0.5016 \sin (\omega s - 60^{\circ}) = 0.659$

$b = 0.6609 - 0.4767 \sin (\omega s - 60^{\circ}) = 0.423$ $H_o H_g = 1.531$

$H_o = 1\pi \times 1.367 \times 3600 \times (1 + 0.033 \cos 360 / 365 \times n) \times (\omega s \sin \delta \sin \phi + \cos \phi \cos \delta \sin \omega s)$

$H_o = 0.428 \text{ KW/m}^2$

$H_g = 0.280 \text{ KW/m}^2$

$H_d = H_g [1.411 - 1.696 H_g H_o] = 0.0844 \text{ KW/m}^2$

$I_d = H_d \times (I_o H_o) = 0.265 \text{ KW/m}^2$

$I_b = I_g - I_d = 0.685 \text{ KW/m}^2$

$I_T = I_{brb} + I_{drd} + I_{grr} = 954 \text{ W/m}^2$

Beam radiation: $\cos \theta = 0.968$

$\theta_1 = 9^{\circ}35'$

Reflective index = 1.52

Angle of refraction $\theta_2 = \sin^{-1}(\sin \theta_1 / 1.52)$

$\theta_2 = 6^{\circ}17'$

$\rho_1 = (\sin(\theta_2 - \theta_1))^2 / (\sin(\theta_1 + \theta_2))^2 = 0.044$

$\rho_2 = (\tan(\theta_2 - \theta_1))^2 / (\tan(\theta_1 + \theta_2))^2 = 0.041$

$\tau_1 = 1 - \rho_1$ $1 + \rho_1 = 0.916$

$\tau_2 = 1 - \rho_2$ $1 + \rho_2 = 0.921$

$\tau = \tau_1 + \tau_2 = 0.919$ 16

$\tau_a = 0.953$

$\tau = \tau_a \times \tau = 0.876$

$(\tau_a)_b = \tau \times 0.99771 - (1 - 0.99771) \times 0.11 = 0.875$

Diffuse radiation:

Assume angle of incident = 60°

$\theta_2 = 35^{\circ}15'$

Following same procedure as for beam radiation,

$(\tau_a)_d = 0.728$

Incident flux absorbed by absorber plate:

$S = I_{brb}(\tau_a)_b + \{I_{drd} + I_{grr}\} \times (\tau_a)_d$

$S = 796 \text{ W/m}^2$

Table.1. Value of Predicting Solar Radiation

Time (h)	Hg (W/m2)	Hd (W/m2)	Ib (W/m2)	Id (W/m2)	Ig (W/m2)	IT (W/m2)	(τ_a)b	S (W/m2)
9:00	249	106	296	234	530	528	0.853	422
10:00	258	100	447	273	720	720	0.868	587
11:00	275	88.35	621	269	890	893	0.874	742
12:00	280	84.4	685	265	950	954	0.875	796
1:00	286	67.7	747	213	960	967	0.874	814
2:00	326	38.85	805	105	910	784	0.868	784
3:00	362	103	541	229	770	773	0.853	631
4:00	408	116	368	182	550	551	0.792	474
5:00	506	144	193	117	310	310	0.573	195

Determination of Area of Collector:

Assumes,

1. Outlet temperature of collector = 60°C

2. Width of collector = 600mm

3. Mass flow rate = 0.024Kg/s

4. Ambient temperature = 30°C

$Q_u = S \times A_p$

$Q_u = m \times C_p \times \Delta T$

$A_p = m \times C_p \times \Delta T / S$

$A_p = 0.024 \times 1005 \times 30796 = 0.90 \text{ m}^2$

Length = Area/width

$L = 0.90 / 0.6$

$L = 1.5 \text{ m}$

4. Experimental procedure

- Initially the solar radiations are strikes the absorber plate through the glass cover.
- There are two types of radiations comes from sun. They are shorter wavelength and another one is longer wavelength radiations.
- The longer wavelength radiations are reflected back and shorter wavelength radiations are arrested in the collector, because shorter wavelengths have lower intensity. So the absorber plate get heated.
- The working fluid is used in this system is air in forced convection using blower.
- Placing baffles on the absorber plate, it can enhances the heat transfer between absorber plate and working fluid.
- The heat was collected in the solar flat plate collector through the working fluid.
- The gained heat was supplied to the drying chamber.so, thedrying chamber gets the additional heat because the chamber also

receives the solar radiations through the glass cover.

- The waffles are arranged and separated one by one in the drying chamber.
- The solar radiations and gained heat which is collected from the collector falls on the waffles and absorbed by waffles. Finally the waffles are dried.

5. Result and discussion

Performance Testing of Mixed Mode Solar Dryer:

We now consider the performance analysis of solar dryer.

The experiment was performed on March 21st and 23rd, 2019 from 9:00AM to 5:00PM for every one hour temperature.

First Day Measurement:

In the month of March 21st 2019, the maximum outlet temperature of flat-plate solar air heater is 75.1°C at 1:00 PM and in drying chamber 81°C is obtained due to presence of glass at the top of the drying chamber.

Table 2. Performance of Solar Dryer in First Day

Performance calculation:

Time (h)	Temperature(°C)				Heat Gain (Qu) (W)	Heat Loss (Ql) (W)	Efficiency (%)
	Surface of Glass	Absorber Plate	Collector outlet	Drying chamber			
9:00	43	85	48.6	50	156.80	188.23	33
10:00	48	105.6	57.3	62	227.61	252.32	35.13
11:00	55	127.15	65.7	69	292.52	314.14	36.4
12:00	58	134.35	72.8	77	343.94	306.87	40.1
1:00	62	131.27	75.1	81	354.06	311.47	40.7
2:00	62	127.25	73.5	79	344.78	296.22	41.59
3:00	54	113.6	68.6	77	308.54	207.37	44.35
4:00	48	101.25	59.3	64	238.57	148.97	48.12
5:00	38	85.57	46.9	48	142.47	16.96	51.06

$$\text{Heat gain (Qu)} = m \times C_p \times (T_{fo} - T_{fi})$$

$$= 8.388 \times 10^{-3} \times 1005 \times (48.6 - 30)$$

$$(Qu) = 156.80W$$

$$\text{Average heat gain} = 267.70W$$

$$\text{Heat loss (Ql)} = (S \times A_p) - Qu$$

$$Ql = (605 \times 1.46 \times 0.56) - 267.70$$

$$= 226.95W$$

$$\text{Efficiency} = (Qu / (IT \times Ac)) \times 100$$

$$\eta = (267.70 / (720 \times 1.5 \times 0.6)) \times 100$$

$$\eta = 41.31\%$$

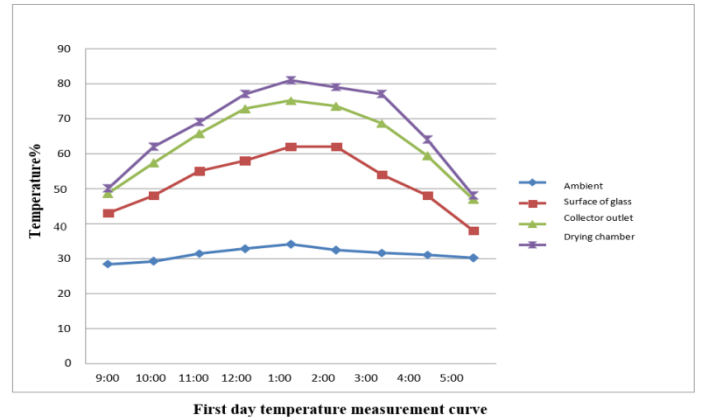


Fig.1. First Day Measurement curve

In the month of March 21st, 2019 At 9:00AM the outlet temperature of air was 48.6°C and in drying chamber 50°C At 1:00PM the outlet temperature of air was 75.1°C and in drying chamber 81°C At 5:00PM the outlet temperature of air was 46.9°C and in drying chamber 48°C.

Second Day Measurement:

The maximum outlet temperature of flat-plate solar air heater is 68.7°C at 1:00 PM and in drying chamber 71°C is obtained due to presence of glass at the top of the drying chamber.

Performance calculation:

$$\text{Heat gain (Qu)} = m \times C_p \times (T_{fo} - T_{fi})$$

$$= 8.388 \times 10^{-3} \times 1005 \times (41 - 29)$$

$$(Qu) = 101.16W$$

$$\text{Average heat gain} = 231.17W$$

$$\text{Heat loss (Ql)} = (S \times A_p) - Qu$$

$$Ql = (605 \times 1.46 \times 0.56) - 231.17 = 263.48W$$

Performance of solar dryer in second day

$$\text{Efficiency} = (Qu / (IT \times Ac)) \times 100$$

$$\eta = (231.17 / (720 \times 1.5 \times 0.6)) \times 100$$

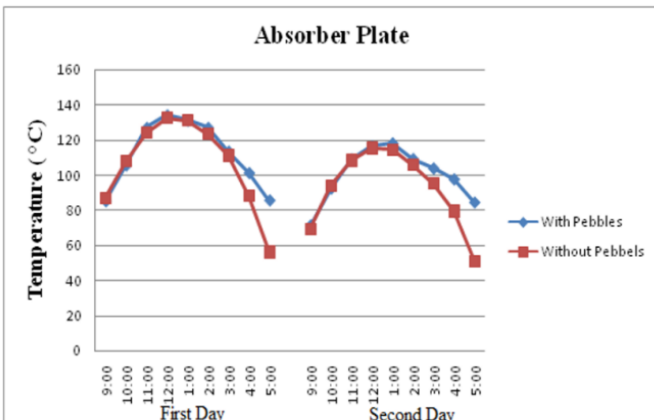
$$\eta = 35.67\%$$

In the month of March 23rd, 2019 At 9:00AM the outlet temperature of air was 41°C and in drying chamber 43°C At 1:00PM the outlet temperature

of air was 68.7°C and in drying chamber 71°C At 5:00PM the outlet temperature of air wa45.2°C and in drying chamber 46°C.

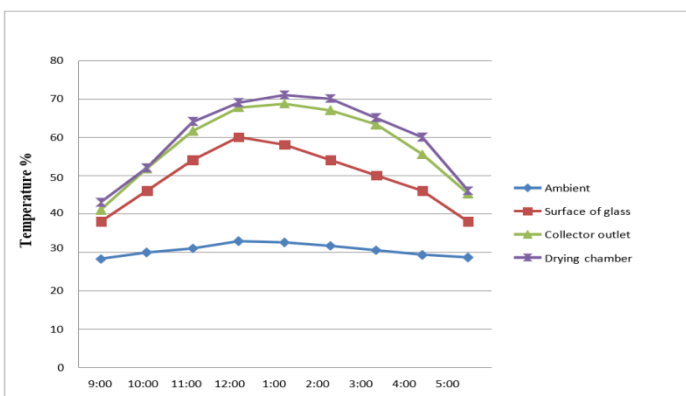
Table.3. Performance of solar dryer in second day

Time (h)	Temperature(°C)				Heat Gain Q_u (W)	Heat Loss Q_l (W)	Efficiency (%)
	Surface of Glass	Absorber Plate	Collector outlet	Drying chamber			
9:00	38	71.21	41	43	101.16	243.87	21.29
10:00	46	92.25	51.8	52	183.77	296.16	28.36
11:00	54	109.26	61.6	64	257.96	348.7	32.1
12:00	60	116.58	67.7	69	300.95	349.86	35.1
1:00	58	118.35	68.7	71	309.38	356.15	35.6
2:00	54	109.36	67	70	303.48	337.52	36.61
3:00	50	103.87	63.3	65	280.72	235.19	40.35
4:00	46	97.65	55.5	60	214.96	172.58	43.35
5:00	38	84.35	45.2	46	128.14	31.29	45.93



Absorber plate temperature with and without pebbles

Fig.2.Second Day Measurement curve



Second Day temperature measurement curve

Fig.3.Comparison of Absorber Plate Surface temperature

Due to presence of pebbles the surface temperature of an absorber plate is much higher than surface temperature without pebbles, because pebbles are sensible heat storing material. So that the efficiency of a solar air heater is much higher in evening session.

Conclusion

From the above design and construction, it has been concluded that solar radiation can be effectively and efficiently utilized by the mixed mode solar dryer for drying the waffles. From the result which has been obtained, the area of the collector (i.e. 0.90m²) is small as portable. The flat-plate solar air heater with storage bed (i.e. pebbles) can be characterized by means of its daily efficiency, determining its daily average characteristics parameter. The average temperature output of solar air heater is 59°C and the efficiency of a dryer is 37%. This temperature is suitable of drying agricultural product also. The time to dry the product is less when compare with open sun drying. Due to presence of pebbles the temperature outlet is much higher when compare to without pebbles during evening session. This project helps us not only increase the temperature efficiency, but also bringing up a new phase in alternative trend in the usage of food processing. It help us to gain lot of technical knowledge which will help in future.

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